Physics-based Integration of H2-Air Rotating Detonation into Gas Turbine Power Plant (HydrogenGT)





Turbine Aerothermal Lab



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Overall aim: develop a high-speed diffuser-turbine to enable integration of rotating detonation combustors (RDC) into industrial turbines

TASKS

- 2. Loss budgeting in a combustor & transition element & turbine 1st stator (nozzle guide vane)
- 4. Scale exp. & comp. studies to F-class and aero-derivative RDE gas turbine integrated system
- 3. Demonstration RDC-transition-NGV coupling towards work production

2. Loss budgeting in a combustor & transition element & turbine 1st stator

- 2.1 Identification of loss mechanisms for the combustor with diffuser and turbine 1st stator
 - Study an existing combustor/diffuser/NGV geometry
 - Computational model validation & predict the combustor losses

2.2. Quantification of the combustor performance metrics with turbine 1st stator

- representative metric of combustor/diffuser/NGV performance
- CARS & PIV data will be used to validate

2.3. Uncertainty quantification of loss mechanisms by integrating high fidelity simulations with the experimental data

• Large Eddy Simulations of the tested optical transparent RDC

Subtask 2.1 Identification of loss mechanisms for the combustor with turbine 1st stator

<u>Goals</u>

- 1. Identify the combustor loss mechanisms from diffuser/turbine 1st stator
- 2. Provide benchmark validation data for high-fidelity simulations

Diagnostic implemented

- High frequency wall-pressure measurements (p_{stat.})
- MHz rate simultaneous orthogonal OH* chemiluminescence and OH-PLIF (χ^{OH*} and χ^{OH})
- Exhaust chemiluminescence (χ^{OH^*})
- Femtosecond Laser Activation and Sensing of Hydroxyl radical (FLASH) velocimetry (v)
- Coherent vortex-velocimetry (v)

Diagnostics being development for this program

- MHz rate high speed optical parametric oscillator
 - OH-PLIF measurements
 - FLASH
 - Coherent vortex-velocimetry
- 100 kHz 1MHz ps-CARS system for
 - Exhaust temperature measurements
- 1 kHz hybrid CARS system (fs/ps CARS)

No NGV installed



Summary of results

- 2.1: Identification of additional loss mechanisms:
 - Azimuthal Reflected Shock Combustion (ARSC) P_{tot}
 Loss mechanism
 - Ramp Combustor has lower performance than BFS
 - Mixing/Deflagrative influences on RDE performance
- 2.2: Total pressure measurements

NGV installed



2.1: Combustor loss mechanism – Comparison of Straight Channel vs Ramp



[1] Athmanathan, Venkat, James Braun, Zachary M. Ayers, Christopher A. Fugger, Austin M. Webb, Mikhail N. Slipchenko, Guillermo Paniagua, Sukesh Roy, and Terrence R. Meyer. "On the effects of reactant stratification and wall curvature in non-premixed rotating detonation combustors." *Combustion and Flame* 240 (2022): 112013.

2.1: URANS Simulations

0.846154 1.49231 M: 0.2 P: 100000 1.17179E+06



Subtask 2.3 Investigation of loss mechanisms in the RDC

High-fidelity Large-eddy Simulation (LES) of THOR H₂/air RDC

- Single detonation wave behavior is observed
- An azimuthal reflected shock combustion (ARSC) wave trails behind the leading detonation wave
- LES depicts similar structures as experiments
- Combustion behavior is further analyzed using species profiles and chemical explosive mode analysis (CEMA)
- Deflagrative combustion is observed in the fill region, varying with the radial location
- Flow profiles at the exit of the transition element





losive mode

(CEM)



T [K]

Subtask 2.3 Uncertainty quantification of loss mechanisms

1.5

[-] M

P_{stat} [-]

0



URANS M: 0.2 0.846154 100000 1.17179E+06 Mach number along the axial direction LES P 5.0 ^{ratio} [-] URANS 0.5 0.02 0 0.03 0.04 0.08 0.05 0.06 0.07 0.09 z [m] 6 ^{×10⁵} Pressure along the axial direction 11 10.5 Pratio [-] 10 LES 9.5 URANS 9 0.02 0.03 0.04 0.05 0.06 0.07 0.08 0.09 z [m]

4. Scale all our studies to F-class and aero-derivative class RDE GT system

4.1. Cycle analysis of the scaled-down RDC-diffuser-turbine

4.2. Cycle analysis to predict the F-class turbine power plant's performance

4.3. Scale lab-scale experimental and computational studies to F-class and aeroderivative class RDEgas turbine integrated systems

Subtask 4.1 Cycle analysis of the scaled-down RDC-diffuser-turbine



3. Demonstration RDC - transition element – turbine 1st stator towards work production

- 3.1. Overall transition element optimization
- 3.2. Computational multi-objective optimization of the turbine 1st stator
- 3.3. Experimental demonstration of the optimal turbine 1st stator @warm conditions
 - Multiple Optimized NGVs characterized in an annular configuration (BRASTA)
 - Inlet Mach number [0.6 1.0]; Reynolds [7.5 *10⁵ 2 * 10⁶]
 - Instrumentation: pressure taps, Kiel & 5H probe traverses, turbulence intensity & oil visualization
- 3.4. Experimental demonstration of the RD combustor and turbine @hot conditions
 - *RDC* + transition element tested first independently, characterization of component performance + turbine inlet flow field.
 - Instrumentation: pressure taps, Kulites, high-frequency Kiel probes, TCs (low & high freq.), high-frequency 3H probe, ALTPs (high-frequency heat flux)
 - *RDC* + transition element integrated with Rolls-Royce M250 turboshaft engine

Subtask 3.1 Overall transition element optimization

FINAL PASSAGE GEOMETRY P_{back} $P_0=7$ bar *Т*₀=1800 <u>К</u> T_total Length allows mixing and shallow slope to prevent separation L = 10 in1700 *T*₀=400 К 1400 1100 800 500 **Exit Profiles** Throat promotes mixing and provides constant 100 100 back pressure to RDC 80 80 Performance vs. back pressure P_{back} P_0 loss 60 60 $P_{0,RDC}$ Exit Peak 'n % span [-] % span [-] Peak T_0 [K] CFD CFD Mach [bar] Mach [kg/s][%] [bar] Nominal -M250 Nominal 5.0 1.48 2.00 1033 40 7.0 0.25 25.5 40 7.0 5.5 1.37 2.00 0.23 1032 18.6 7.0 6.0 1.09 2.00 0.21 1031 11.7 20 20 7.0 6.5 0.60 1.60 0.16 1035 5.56

700

800

900

T0 [K]

1000

1100

Design condition



0

0

0.05

0.1

M [-]

0.15

0.2

Subtask 3.2 Computational multi-objective optimization of the turbine 1st stator

- Optimized NGV design from Subtask 3.2. Design condition M₁=0.6
- Designed for a large operating range, M1 up to 1 (full subsonic operability)
- High turning $(\alpha_2 \simeq 70^\circ) \rightarrow \text{high power}$

Can be pushed to high pressure ratios: $P_{01}/P_2 = 2.1$, $M_2 = 1.03$



Subtask 3.3 Experimental demonstration of the optimal turbine 1st stator

- BRASTA: Pressure-driven Annular Test Section
- 4 geometries to test:
 - Baseline & Optimized (Design case 1)
 - Baseline & Optimized (Design case 2)



- Detailed characterization of NGV flow field
 - Instrumentation: pressure taps, Kiel & 5H probe traverses, turbulence intensity & oil visualization
- Extensive testing envelope
 - Inlet Mach number [0.6 1.0]; Reynolds [7.5*10⁵ 2* 10⁶]



Subtask 3.4 Experimental demonstration of RDC + turbine



Subtask 3.4 Experimental demonstration of RDC + turbine

Overview of modifications: Cooled RDC Integration with M250

- Fresh mixture injected from tanks not from compressor discharge (engine loop not closed)
- Compressor acts as HPT brake (compressor outlet is throttled)
- LPT connected to generator + load bank for power dissipation

