# High-temp Dry Gas Seals for sCO<sub>2</sub> turbines DE-FE0031924

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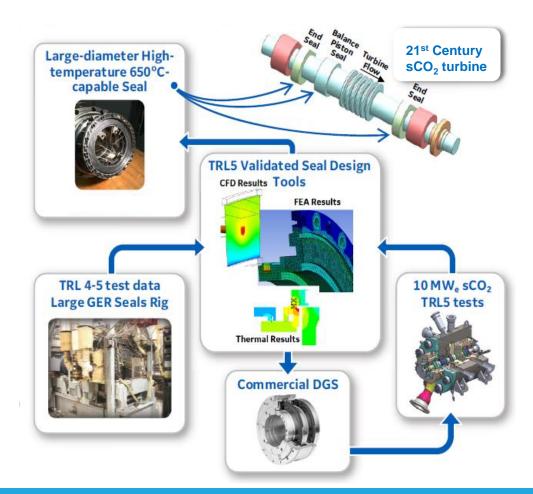
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# Outline

- Overview
- Commercial DGS Failures & Need for High-temp DGS
- Thermal Modeling and DGS testing in sCO2 compressors Task 2
- Large-diameter seal design Task 3
- Summary & Next Steps

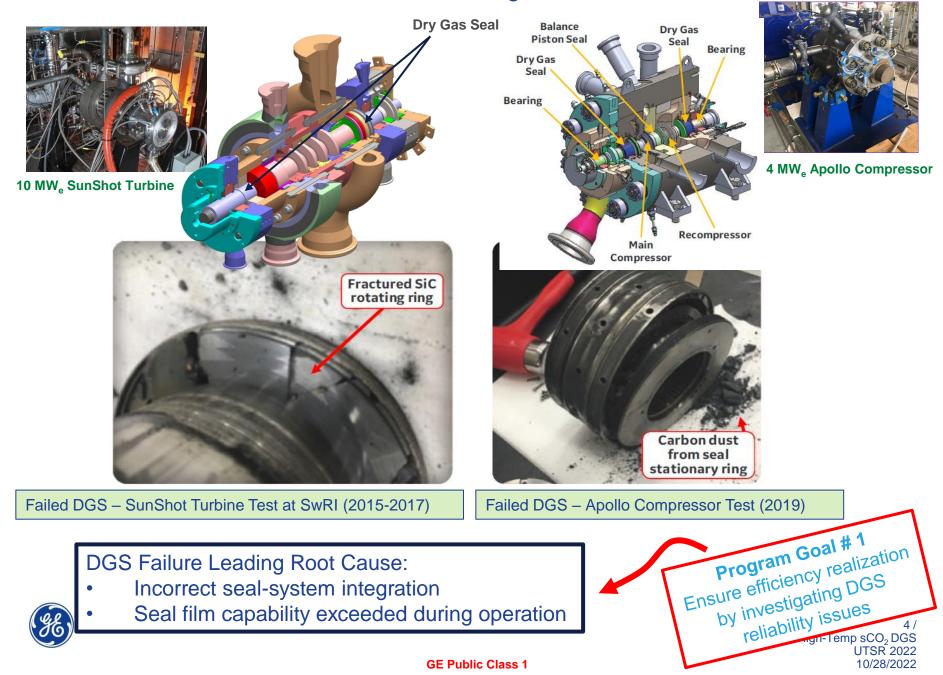


### Overview



- 10 MW<sub>e</sub> scale Improved reliability of DGS and validation of Seal Design Tools
- 100 MW<sub>e</sub> scale High-temp seal to enable modular sCO<sub>2</sub> 21<sup>st</sup> century turbines

#### DGS Reliability Issue – 10 MW<sub>e</sub> scale



# Value of High-temp DGS Development Ensure efficiency realization

- State-of-the-art low-temp (200 °C) commercial DGS
  - 0.5% to 1.3% cycle efficiency
  - Limit operating ramp rates of turbines
- High-temp DGS (up to 700 °C)
  - Enable 0.7% to 1.9% cycle efficiency
  - Enable alternative turbine architectures
- Enable even better er in in better turbine ramp rates & operation flexibilit
  - Higher ramp rates

Turbomachinery	Application	Location and Temperature	Cycle Efficiency Benefit
10 MW <sub>e</sub> sCO <sub>2</sub> turbine	WHR, CSP, NGPC, Coal FIRST risk reduction	Shaft-end labyrinth seals (200°C)	0 (baseline)
		Shaft-end DGS (200°C)	1.32%
		Shaft-end and Balance-piston DGS (700°C)	1.96%
10 MW <sub>e</sub> sCO <sub>2</sub> com- pressor	WHR, CSP, NGPC, Coal FIRST risk reduction	Shaft-end labyrinth seals (200°C)	0 (baseline)
		Shaft-end DGS (200°C)	1.32%
		Shaft-end and Balance-piston DGS (200°C)	1.9%
100 MW <sub>e</sub> sCO <sub>2</sub> turbine	Coal FIRST	Shaft-end labyrinth seals (200°C)	0 (baseline)
		Shaft-end DGS (200°C)	0.71%
		Shaft-end and Balance-piston DGS (700°C)	0.85%
450 MW <sub>e</sub> sCO <sub>2</sub> turbine	Fossil indirect coal plant	Shaft-end labyrinth seals (200°C)	0 (baseline)
		Shaft-end DGS (200°C)	0.52%

#### **GE Public Class 1**

#### Goal # 1 Reliability

by increasing reliability

Maure even vour flexible) 8 operationally the vice Coal FIRST turbines

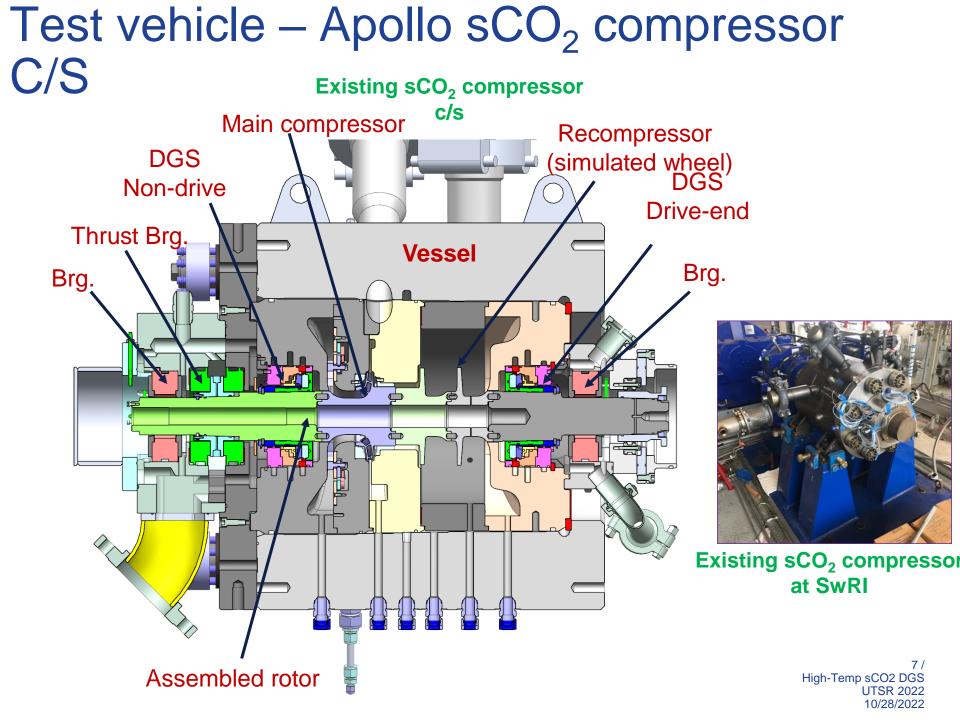
#### Goal # 2 Higher -Temps

High-Temp sCO<sub>2</sub> DGS TSR 2022 10/28/2022

### Update on Task 2

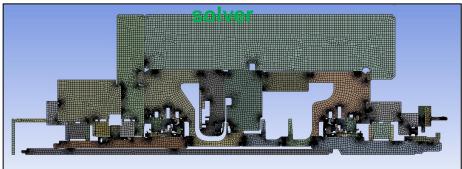
## Thermal modeling & testing of instrumented DGS in sCO2 compressor



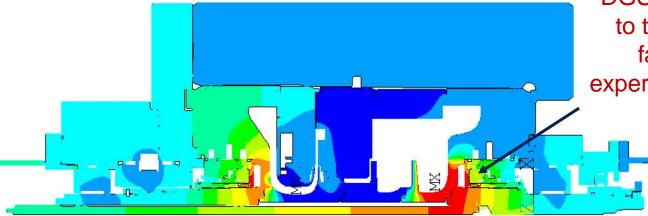


## Thermal Modeling of DGS in Compressor

In-house FE-based thermal



**Temperature prediction for** 

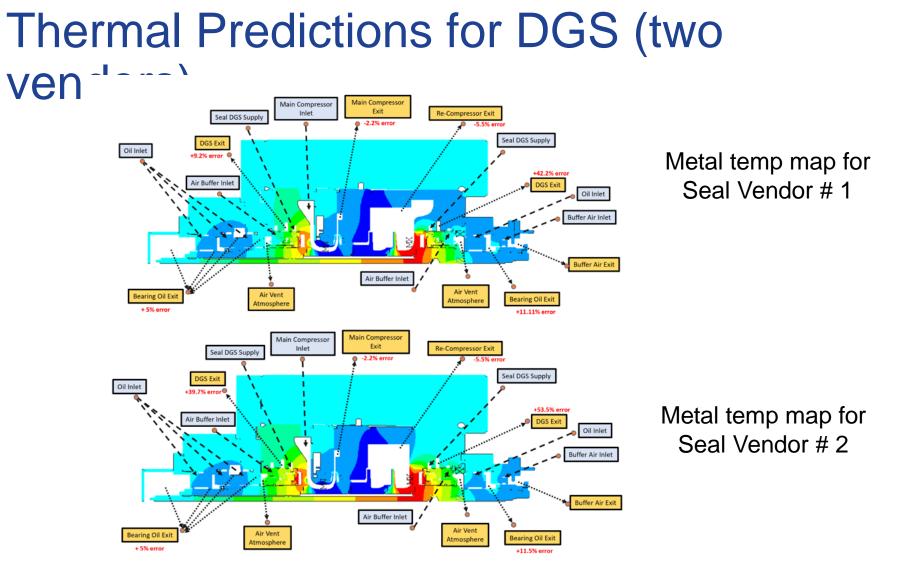


High temperatures in DGS region can lead to thermal stability failure -- need experimental validation

- In-house thermal solver accounts for CO2 real-gas properties, windage, CFD-based leakage flows, heat transfer coefficients
- High temperatures (and large thermal deformation) predicted near Drive-End DGS due to a combination of large heat generation and low cooling flow



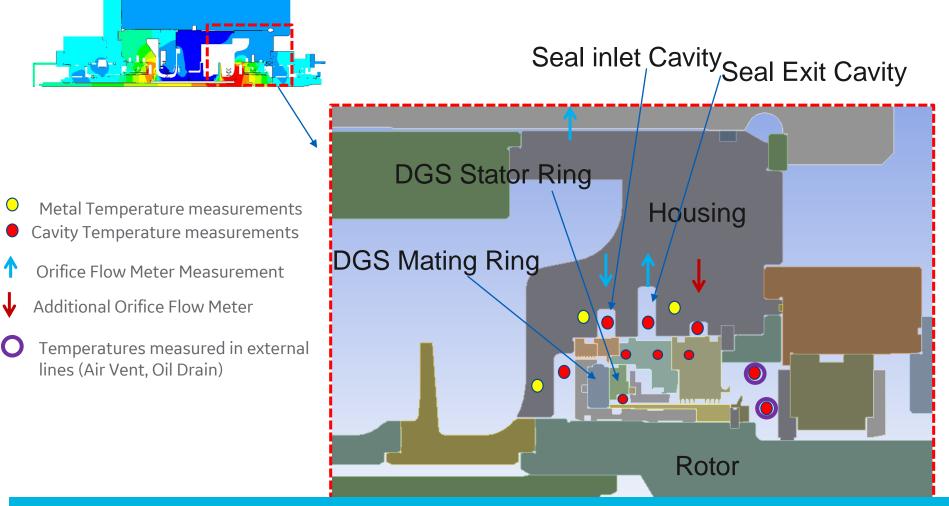
imagination at work



Thermal predictions completed for seals from two different commercial DGS



#### DGS Instrumentation scheme for upcoming testing



- Devised cavity and metal temperature measurement scheme to validate thermal predictions
- Worked with commercial DGS vendor to procure DGS with custom instrumentation access
- Worked with in-house machine shop to fabricate new housing with instrumentation access



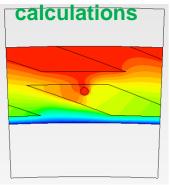
### Update on Task 3

## Thermal modeling & testing of instrumented DGS in sCO2 compressor

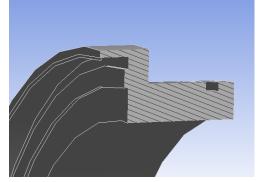


#### Large-diameter Seal design – ongoing work

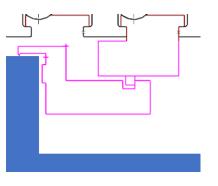
#### Seal bearing face CFD for force balance & leakage



Seal c/s analysis with FEA for stresses & deflections



Seal packaging in a large land-based turbine to study assembly and space constraints





• Partnering with GE Vernova (Power) team to develop a seal design for a land-based turbine

# Summary & Next Steps



# Summary & Next Steps

- Task 2
  - Thermal analysis & pre-test predictions ready
  - Testing commences Jan 2023 with instrumented DGS in sCO<sub>2</sub> compressor
- Task 3
  - Large-diameter, high-temp seal design for a landbased Steam turbine underway

