# Combustion Modeling for Direct Fired Supercritical CO2 Power Cycles

SBIR Phase II Review Meeting Award # DE-SC0017235 May 11<sup>th</sup>, 2021

Presented by

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Acknowledgment: This material is based upon work supported by the Department of Energy Award Number DE-SC0017235.

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### **Project Description and Objectives**

#### **Project Purpose:**

- Develop a tractable and accurate turbulent combustion modeling approach that may be applied to design trade studies of direct fired supercritical CO<sub>2</sub> power cycle combustors
- Supports DOE objective of improving the efficiency of large scale fossil energy based power generation systems
- Enhanced accuracy of a validated combustion model will enable design optimization for efficiency
- Current combustion modeling approaches not validated for supercritical flows

#### Approach:

- Develop "direct quadrature method of moments" (DQMOM) combustion model for low-speed,
  supercritical flows
- Validate the model with new experimental data

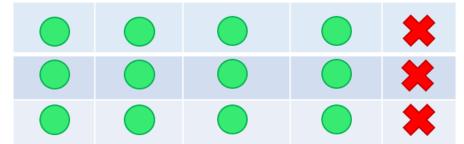




### **Turbulent Combustion Model Technology Comparison**

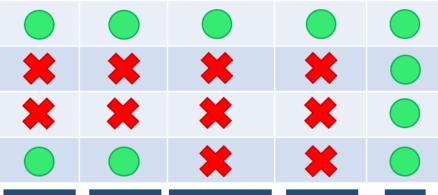
#### **Stochastic Models:**

- PDF transport
- Linear-eddy model (LEM)
- One dimensional turbulent (ODT)



#### **Moment Models:**

- DQMOM
- Flamelet progress variable (FPV)
- Conditional moment closure
- LEM counter flow (LEM-CF)











tractable





### Direct Quadrature Method of Moments (DQMOM)

#### **Foundation:**

- Direct solution of the scalar PDF transport equation
- Eulerian solution procedure with discretized PDF
- Employs validated PDF mixing models

#### **Advantages:**

- Applicable to low and high speed flows
- Generalize boundary condition treatment
- Comparable accuracy to velocity scalar stochastic PDF model
- Captures localized flame extinction
- Tractable for routine design application





### Phase II Project Description

#### **Model Development Tasks:**

- Task 1.1: DQMOM Preconditioned (Low Speed) Formulation Extension
- Task 1.2: Real Fluid Chemical Kinetic Effects Extension
- Task 2.1: Detailed Chemical Model Development
- Task 2.2: Reduced Chemical Model Development

#### **Model Validation Tasks:**

- Task 3.1: SwRI Bench-Top Combustor Validation
- Task 3.2 3.4: SwRI 1 MW Combustor Validation





### Task 3.1 Objective:

- Validate the DQMOM combustion model for application to a laboratory scale sCO<sub>2</sub>
  combustor configuration
- Validation data generated by SwRI

#### **SwRI Bench-Top Combustor:**

- SwRI has designed an new non-premixed laboratory scale combustor for this validation effort
- Co-axial jet configuration
- Will provide excellent validation data for combustion model development





#### **Test Objectives:**

- Establish the ignition limits of the combustor
- Provide wall pressure and temperature measurements for a stable flame condition

#### **Ignition Limit Determination:**

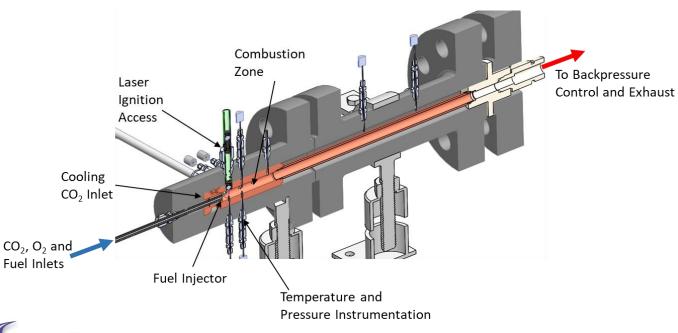
- Laser ignition of the flow occurs in the downstream jet shear layer
- Establishment of a stable flame will require the upstream propagation of the ignition front in the partially premixed shear layer up to the jet lip recirculation zone
- A series of cases varying the global O/F ratio used to established the ignition limits
- Ignition visualized through optical ports
- Possible lifted flames over a narrow O/F ratio range





### **Rig Overview:**

- Small, ~0.5 in diameter direct fired combustor
- Non-premixed, coaxial jet, fuel injector







#### Instrumentation and controls:

Instrumentation and DAQ installed, fuel cart assembled





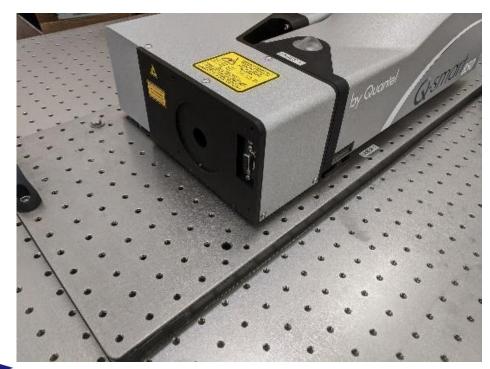






### **Laser Ignition and Optical Probes:**

- Quantel Q-smart Twins 2W laser with water-cooled probe for ignition
- Water-cooled optical probes









#### **Exhaust:**

- Backpressure valve to maintain desired combustor pressures
- 13' parallel flow water-cooled heat exchanger



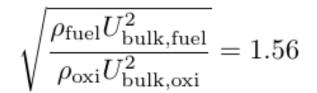


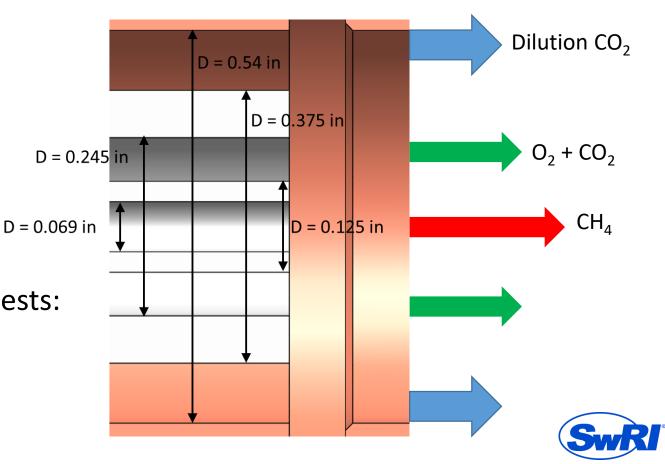




### **Anticipated Test Conditions:**

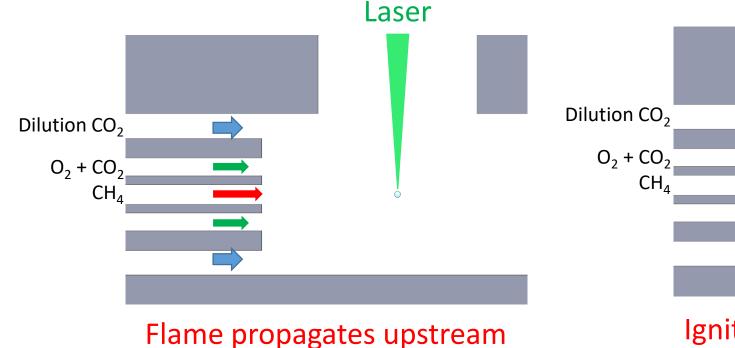
- Change in sCO<sub>2</sub> source necessitated new operating conditions
- Inlet sCO<sub>2</sub> pressure: 1600psi
- Inlet sCO<sub>2</sub> temperature: 200F
- ~45 tests to determine flammability
- Injector bevel removed
- Momentum flux ratio constant for all tests:



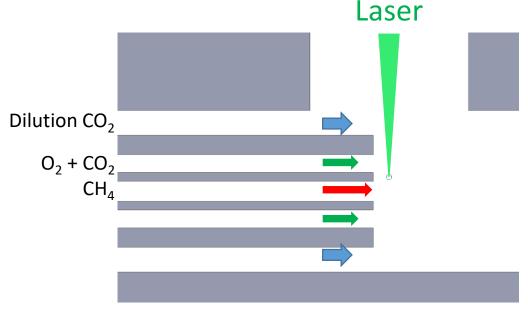




### **Ignition locations:**



(installed)



Ignition in recirculation region (backup option)





### Testing Delays:

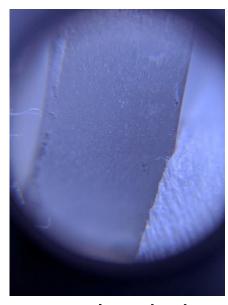
- sCO<sub>2</sub> sourcing difficulties: Delays of other SwRI projects tied up pumps and heaters,
  and denied access to test facilities during extended test campaign
- Staffing change: Dr. Jacob Delimont replaced by Dr. Brian Connolly
- COVID-19: SwRI personnel sick and/or quarantined for multiple weeks, as well as longer supplier lead times



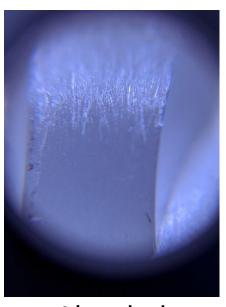


#### **Current Status:**

- Rig and fuel cart are assembled
- sCO<sub>2</sub> tank and heater reserved, control program written
- Plumbing and DAQ networking being completed
- Water-cooled laser probe assembled
- Abrading sapphire lenses for adhesive contact
- Final laser probe modifications ongoing
- Light-off imminent!



Unabraded



**Abraded** 





#### Test Schedule:

- Commissioning to occur end of April or early May
- Data collection to begin in May
- This project has priority for all equipment and personnel



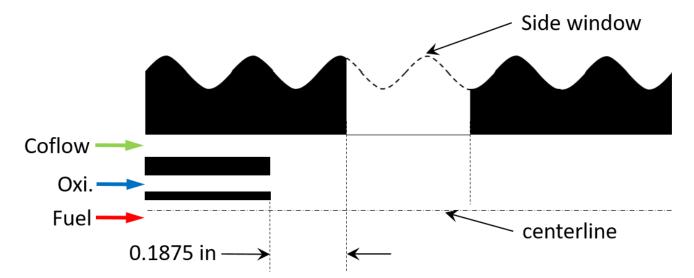


#### **Model Application Strategy for Validation:**

- Apply unsteady, 2-D/axisymmetric RANS analysis to bracket ignition limit
- Refine ignition limit with 3-D LES analysis

#### Combustor Analysis Schematic (to scale):

- Laser ignition in the center of the side window
- Flame must propagate upstream to the base recirculation zone







### **Pretest Analysis:**

- Unsteady, 2-D/axi RANS analysis of preliminary test conditions from Fall 2020
- Laser ignition modeled with 2 mm diameter energy source term to raise local temperature to ~ 3000 K
- Simulate unsteady combustor start-up, laser ignition transient, and flame development
- Test matrix generated for:

$$-P_0 = 1.3e + 7 Pa$$

- Maintaining constant bulk fuel - oxidizer mass flux ratio

Feed Stream	Composition	Max. $\dot{m}$ (kg/sec)	Temperature (K)
Fuel	CH <sub>4</sub>	0.004	473.15
Oxidizer	$O_2+CO_2$	O <sub>2</sub> : 0.016, CO <sub>2</sub> : 0.04	473.15
Coflow	$CO_2$	0.2	473.15





### **Pretest Analysis:**

Fuel and coflow/dilution stream conditions:

98	Feed Stream	Composition	<i>m</i> (kg/sec)	Ubulk (m/sec)	Temperature (K)
33-	Fuel	CH <sub>4</sub>	0.0025	15.0	473.15
	Coflow	$CO_2$	0.05	3.43	473.15

Oxidizer stream conditions:

Mass fraction					
Case no.	O <sub>2</sub>	CO <sub>2</sub>	m (kg/sec)	Ubulk (m/sec)	Temperature (K)
1	1.0	0.0	0.017915	6.926	473.15
2	0.75	0.25	0.01887	6.573	473.15
3	0.625	0.375	0.01941	6.389	473.15
4	0.5	0.5	0.02	6.200	473.15
5	0.4	0.6	0.02052	6.048	473.15
6	0.3	0.7	0.02107	5.887	473.15





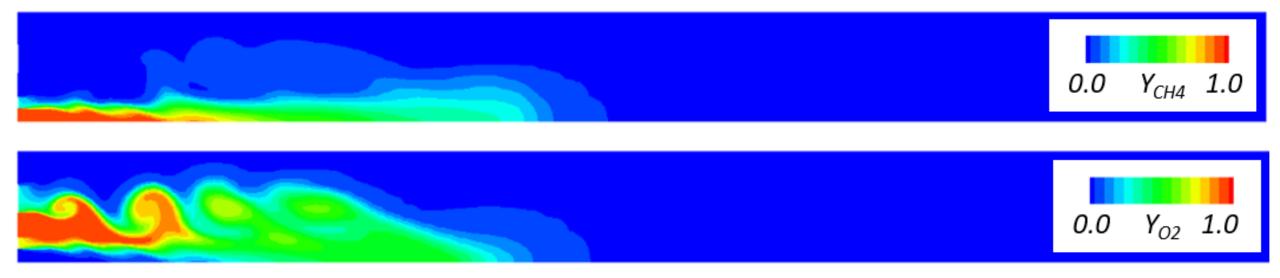
#### **Simulation Procedure:**

- Simulation initialized with cold coflow fluid (CO<sub>2</sub>) throughout the domain
- Fuel and oxidizer inflow transient simulated *nonreacting from t* = 0 to t = 0.01 sec:
  - Fuel oxidizer shear layer sets up
  - Mixing region reaches past the laser ignition location
- Reacting flow simulation with laser ignition source from t = 0.01 to t = 0.014 sec:
  - Laser source a 2 mm diameter spot at the upstream edge of the viewing window
  - Energy source term raises the local minimum temperature to  $\sim$  3000 K
  - Flame anchored to the laser ignition location
  - Flame may propagate upstream to the fuel oxidizer base recirculation zone
- Laser source removed and *reacting* flow simulation continued from t = 0.014 to  $t \sim 0.032$  sec:
  - Flames below the ignition limit blow out
  - Stable flames anchor to the base fuel oxidizer base recirculation zone



Case 1 Predictions (oxidizer flow  $-Y_{02} = 1.0$ ):

$$t = 0.006 sec$$

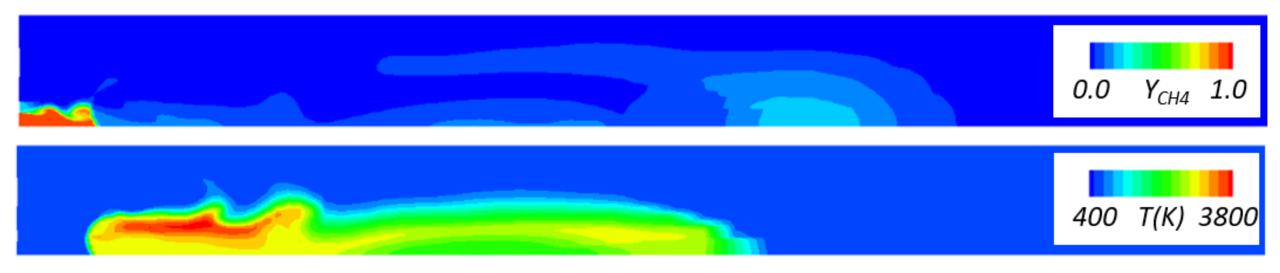






Case 1 Predictions (oxidizer flow  $-Y_{02} = 1.0$ ):

$$t = 0.0114 sec$$

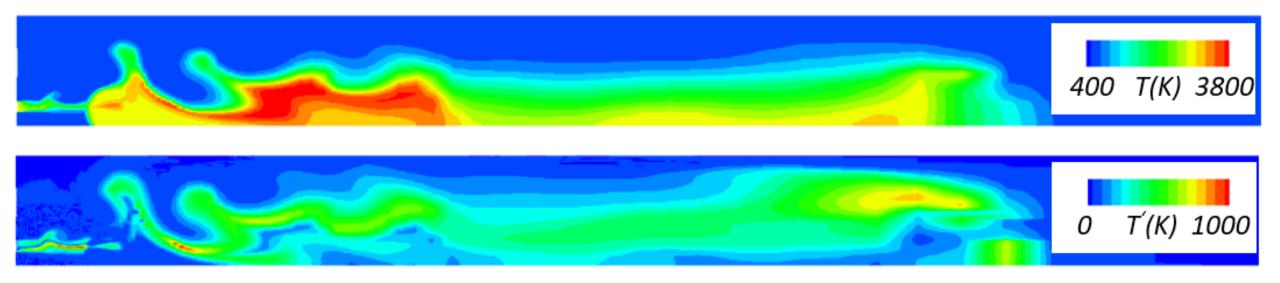






Case 1 Predictions (oxidizer flow  $-Y_{02} = 1.0$ ):

$$t = 0.0114 sec$$

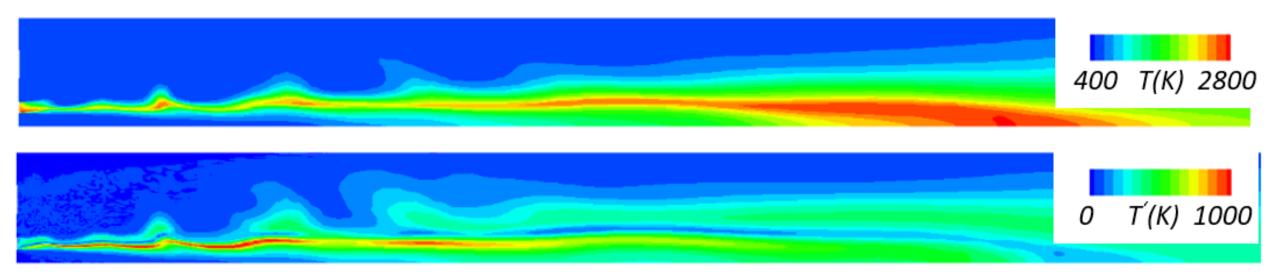






Case 1 Predictions (oxidizer flow –  $Y_{02} = 1.0$ ):

$$t = 0.031 sec$$







### **Preliminary Pretest Ignition Results:**

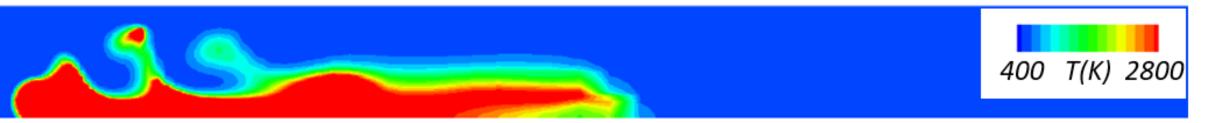
Case no.	O <sub>2</sub> Mass Fraction	Ignition/Stabilization Status
1	1.0	anchored flame
2	0.75	anchored flame
3	0.625	anchored flame
4	0.5	flame blow-out
5	0.4	flame blow-out
6	0.3	flame blow-out



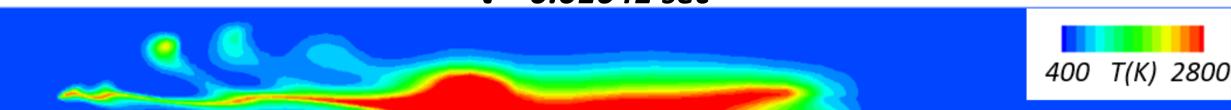


Case 4 Predictions (oxidizer flow –  $Y_{02} = 0.5$ ):

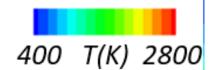
t = 0.0147 sec



t = 0.01641 sec



t = 0.0431 sec



### **Current Status**

#### Task 3.1:

- 3-D LES grid generated for the combustor ( ~ 2.3m points)
- Computational efficiency enhancements implemented:
  - Decoupled scalar solution procedure ~ 50% cpu time reduction
  - Automated parallel ODE chemistry procedure ~ 50% cpu time reduction
- Waiting on final test conditions from SwRI

#### Task 3.2 – 3.4:

- 3-D LES grid generated for January 2021 combustor geometry ( ~ 2.3m points)
- Initial testing for preliminary design operating condition
- Waiting on final geometry and test conditions for SwRI





### **Preparing Project for Next Steps**

### **Project Market Benefits/Assessment:**

- Validated and tractable simulation tool developed for sCO<sub>2</sub> direct fired combustor design parametric studies
- Ignition data generated for validation of other modeling approaches
- Applicable to rocket engine design and analysis

#### <u>Technology-to-Market Path:</u>

- Project technology implemented with CRAFT Tech commercial software products:
  - CRAFT CFD®, RPFM® and RNFM®
- Validation with data remains to be completed due to testing delays
- New research direction parameterized chemistry for efficiency enhancement
- Continued collaboration with SwRI critical to the validation effort





# **Concluding Remarks**

- DQMOM turbulent combustion model provides a tractable avenue for accurate design and analysis prediction of sCO<sub>2</sub> direct fired applications
- DQMOM formulation:
  - Provides for a *predicted* scalar PDF within the flow
  - Applicable to generalized boundary conditions characteristic of real combustors
- Model development efforts "complete":
  - Refinements may be required as validation effort continues
- Model validation effort slowed by testing delays:
  - Bench-top combustor data collection to begin in May
  - 1 MW combustor data collection to begin at the end of 2021



